# **EFFECT OF SPIRAL GROOVES IN PISTON BOWL ON EXHAUST EMISSIONS OF DIRECT INJECTION DIESEL ENGINE**

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#### Abstract

In present time developed and developing country are used from small to largest diesel engine as a power plant for different purpose like generating electricity and transportation. These engines consume in heavy quantity fuel per hour, where these engines produce the bulk power as well as there also produce the different types of toxic gases that harmful our human beings and environment. At this time both type of country try to reduce the harmful gases from diesel engine In present scenario many technology are used to reduce the toxicity of exhaust gases EGR (Exhaust Gas Recirculation) is one of them. Where EGR reduces the toxicity of exhaust gases and fuel consumption as well as their reduce the power of engine which is not good at any level. In this present experimental work to reduce the NOx, HC, CO, and CO<sub>2</sub> some modification has been done in piston bowl by cutting three spiral grooves on inner surface of hemispherical bowl and slight increasing in bowl diameter. The spiral grooves increase the air capacity and slight reduce the compression ratio as well as make homogeneous mixing of air and fuel. This experiment is done on Kirloskar AV1 water cooled, natural aspirated direct injection diesel engine with pure diesel. In experiment it is observed the fuel consumption and NOx reduction by 0.1Kg per hour 8.82%. respectively.

Keywords: D.I.Diesel Engine, Spiral Grooved Piston, Swirl, Emissions

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# **1. INTRODUCTION**

Environmental pollution at this time is very serious problem for our human beings and flora-fauna. Our environment is polluted day by day from industrial emissions and road vehicles emissions. Petrol engine and diesel engine produced different types of harmful gases during combustion like NOx, CO, CO<sub>2</sub>, HC and some quantity SOx due to poor fuel quality. These gases are produced by different engine factor such as piston bowl geometry ,injection timing, compression ratio etc. All these factor also affects the combustion efficiency, fuel consumption and engine brake power.

To reduce the emissions engine manufacturers try to best design the combustion chamber and other level. At combustion chamber geometry design to reduce the NOx many researchers studied the different piston bowl geometry. Cao Li et al reported the NO and CO emission are lower for the vertical wall bowl shape and Re- entry bowl shape as comparison to open bowl shape [14]. In numerical study of NOx reduction Liu found the NO can be reduced by changing the bowl depth [13]. By theoretical investigation Risi.A.De. et al also reported lower NOx emission for different piston bowl shape at higher speed [12]. In experimental study of tangential grooves on piston crown by Reddy .C.V et al also reported the lower CO emission for different fuels[11].

So, in current scenario every researcher and manufacturers try to reduce the exhaust emissions by combustion chamber design as well as other method like EGR (Exhaust Gas Recirculation), SCR (Selective Catalytic Reactor) and HCCI ( Homogeneous Charge Compression Ignition). Here in present work NOx and other gases are reduced by the modification of piston bowl geometry for direct injection diesel engine.

# 2. EXPERIMENTAL SETUP

To measure the NOx and other exhaust gases like CO,CO<sub>2</sub> and HC the experiments are conducted on single cylinder four stroke water cooled direct injection diesel engine Kirloskar AV1 engine .To measure the exhaust gases ARO, Quattro Pro GA-4050 five gas analyser is used. Detail specifications of engine setup and gas analyser are given in table (1) and table (2).

# 2.1 Specifications of Diesel Engine and Rope Brake Dynamometer





Table 1

Item	Specification	
Kirloskar Diesel Engine	AV1	
Model		
Engine power	3.7 KW	
Cylinder Bore	80 mm	
Stroke Length	11 0mm	
Engine Speed	1500 rpm	
Compression Ratio	16.5 : 1	
Swept volume	553 cc	
Stroke	Four	
Injection Pressure	190 bar	
Injection Timing	27 <sup>°</sup> b TDC	
Number of hole of injector	Three x 0.25 mm	
and size		
Dynamometer	Rope Brake Type	
Diameter of Brake Pulley	30 mm	

#### 2.2 Specifications of Gas Analyser

Table 2

ARO , Quattro Pro GA-4050 Five gas Analyzer					
Gases	Specified Range	Accuracy Volume	Accuracy	Resolution	
СО	0-10 %	0.06 %	3%	0.01 %	
$CO_2$	0-20 %	0.4 %	4 %	0.1 %	
HC	0-20000 ppm	12 ppm	5 %	1 ppm	
0	0-21 %	0.1 %	3 %	0.01 %	
NO	0-5000 ppm	25 ppm	5 %	1 ppm	

# 2.3 Modified Piston

In modified piston three spiral grooves has been made on inner surface of hemispherical bowl piston and slight increasing in piston bowl diameter .The spiral grooves make homogeneous mixing of air and fuel during combustion by enhancing the swirl within combustion chamber and increased bowl diameter decrease the compression ratio and its effect on emission reduction are observed. In figure (2) both piston are presented with section view.



Standard Piston

Modified Piston



Fig 2 Piston Section View

Meshing of Piston

# **2.4 Experimental Procedure**

The experiment are performed on D.I .Diesel Engine at constant speed 1500 rpm with injection pressure 190 bar by using pure diesel. In first phase the data recorded with standard piston (hemispherical bowl shape) and in second phase the data recorded by changing the modified piston (spiral grooved piston). The power of engine is measured by the rope brake dynamometer that is coupled with engine and engine exhaust emission are measured by ARO five gas analyser at different load. The performance and emission characteristic are compared with standard piston results.

#### **3. RESULTS AND DISCUSSION**

#### 3.1 Brake Thermal Efficiency

Since brake thermal efficiency of engine depends on different engine factors like engine piston bowl shape. Here in modified piston the spiral grooves make homogeneous mixing of air and fuel that amplify the combustion efficiency .In figure 3 the comparison is shown with respect to brake power of engine. In this experiment we the 24.3% for the standard piston and 26.2 % for modified piston, in this way at full load of engine the increment in brake thermal efficiency is 1.9 %.



Fig.3 Brake Power vs Brake Thermal Efficiency

#### 3.2 Brake Specific Fuel Consumption

In figure 4 the comparison of BSFC for the two pistons with respect to engine brake power are presented. From figure it can be observed that as the engine load increases the BSFC reduces respectively. But for modified piston the BSFC slightly more reduced at full load than Standard Piston. The BSFC reducing showed the lower fuel consumption that is fuel economy.



Fig.4 Brake Power vs BSFC

#### 3.3 Nitrogen Oxides

NOx emission from engine depends on various factor like compression ratio, temperature, piston bowl shape, and injection pressure etc. Here in modified piston the spiral grooves slightly reduced the compression ratio that regulates the peak temperature of combustion and homogeneous mixing help to decreasing the NOx during the combustion. It is observed that the NOx is reduced by 8.82 % at full engine load.



Fig.5 Brake Power vs NO<sub>x</sub>

#### 3.4 Hydrocarbon Emission

Hydrocarbon emission reduction at full load significantly not much lower than standard piston. However it is reduced by 4.6 % for modified piston.



Fig.6 Brake Power vs HC

#### 3.5 Carbon Monoxide

The reduction in carbon monoxide for modified piston is observed near about 7.7 % than standard piston. The CO reduction is not more as many possible but spiral grooves helps better mixing of air and fuel during combustion.



Fig.7 Brake Power vs CO

#### **3.6 Exhaust Gas Temperature**



Fig .8 Brake power vs Exhaust Gas Temperature

During experiment it observed that as the engine load increasing the exhaust gas temperature also increases. For standard piston the maximum exhaust gas temperature is  $520^{\circ}$ C at full load and for modified piston the maximum exhaust gas temperature is  $495^{\circ}$ C which is lower than standard piston . The spiral grooves affects turbulence and swirl within combustion chamber and cause the reduction of gas temperature that is lower cooling requirement and safe operation as well as NOx reduction. In figure 10.7 the comparison is shown for both piston and temperature reduction by 4.8 %.

#### 4. CONCLUSIONS

It is observed from experiments as the engine load increases

1- The fuel consumption slightly reduced by 0.1 Kg per hour.

2- Brake thermal efficiency for modified piston is increased near about 1.9 & 2%.

- 3- Brake specific Fuel Consumption is also reduced.
- 4- NOx is reduced by 8.82 % at full engine Load.
- 5- HC is reduced but not significantly as many possible.
- 6- CO is also reduced but not much more as expected.
- 7- The temperature of exhaust gas is reduced by 4.8 %.

# FUTURE SCOPE

Spiral grooves in piston bowl enhances the air swirl within combustion chamber as well as increases the air capacityduring suction and slightly reduces the compression ratio that controlled the peak temperature during combustion. By this way fuel consumption and NOx are reduced. However, further investigations are required for turbulence during compression stroke, HCCI combustion and injection parameters like injection timing and injection pressure to improve the present work.

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